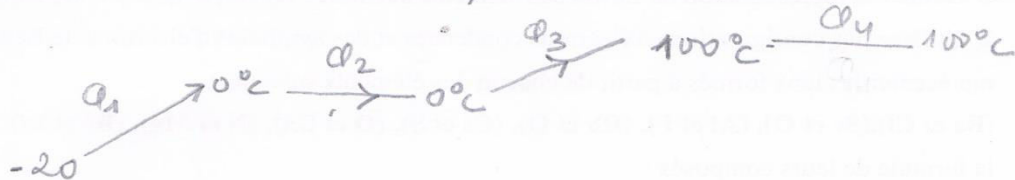


Exercice 1:

I) la glace se réchauffe

$$Q_1 = m_{\text{glace}} C_{p\text{glace}} (T_0 - T_{-20}) = 10 \times 2,1 (0 + 20) = 420 \text{ J} =$$



Q_2 : quantité de chaleur pour fondre la glace

$$Q_2 = m L_f = 10 \times 334,4 = 3344 \text{ J}$$

$$Q_3 = m_{\text{eau}} C_{p\text{eau}} (T_{100} - T_0) = 10 \times 4,18 (100) = 4180 \text{ J}$$

$$Q_4 = n L_v = \frac{10}{18} \times 40500 = 22500 \text{ J}$$

$$Q_T = Q_1 + Q_2 + Q_3 + Q_4 = 420 + 3344 + 4180 + 22500 = 30444 \text{ J}$$

II) $\sum Q_i = 0 \Rightarrow Q_{\text{cal}} + Q_{\text{eau}} = 4375$

$$1) m_{\text{H}_2\text{O}} C_{p\text{H}_2\text{O}} (T_{e1} - T_1) + m_{\text{H}_2\text{O}} C_{p\text{H}_2\text{O}} (T_{e1} - T_1) = 4375$$

$$m_{\text{H}_2\text{O}} = \frac{4375 - m_{\text{H}_2\text{O}} C_{p\text{H}_2\text{O}} (T_{e1} - T_1)}{C_{p\text{H}_2\text{O}} (T_{e1} - T_1)} =$$

$$m_{\text{H}_2\text{O}} = \frac{4375 - 50 \times 1 (45 - 20)}{1 \times (45 - 20)} = 125 \text{ g}$$

2) C_p de l'air

$$\sum Q_i = 0 \Rightarrow Q_{\text{cal}} + Q_{\text{H}_2\text{O}} + Q_{\text{air}} = 0$$

$$Q_{\text{air}} = -(Q_{\text{cal}} + Q_{\text{H}_2\text{O}}) = -4375$$

$$Q_{\text{air}} = m_{\text{air}} C_{\text{air}} (T_{e1} - T_1) = -4375$$

$$C_{\text{air}} = \frac{-4375}{m_{\text{air}} (T_{e1} - T_1)} = \frac{-4375}{50 \times (45 - 90)} = 1,94 \frac{\text{cal}}{\text{g}^\circ\text{C}}$$

* Calcul de la chaleur latente de fusion de la glace.

$$Q_{\text{cal}} + Q_{\text{H}_2\text{O}} + Q_{\text{air}} + Q_{\text{glace}} = 0$$

$$Q_{\text{glace}} = m_g L_f + m_g C_{\text{H}_2\text{O}} (T_{e2} - T_f)$$

$$\Rightarrow \mu_{\text{H}_2\text{O}} C_{\text{H}_2\text{O}} (T_{e2} - T_{e1}) + m_{\text{H}_2\text{O}} C_{\text{H}_2\text{O}} (T_{e2} - T_{e1}) + m_{\text{air}} C_{\text{air}} (T_{e2} - T_{e1}) + m_g L_f + C_{\text{glace}} m_{\text{glace}} (T_{e2} - T_f) = 0$$

$$\Rightarrow L_f = \frac{-(T_{e2} - T_{e1}) [\mu_{\text{H}_2\text{O}} C_{\text{H}_2\text{O}} + m_{\text{H}_2\text{O}} C_{\text{H}_2\text{O}} + m_{\text{air}} C_{\text{air}}] - m_g C_{\text{glace}} (T_{e2} - T_f)}{m_g}$$

$$\Rightarrow L_f = \frac{-(11,39 - 45) (125 \times 1 + 50 \times 1 + 50 \times 1,94) + 100 \times 1 (11,39 - 0)}{100}$$

$$L_f = 80 \frac{\text{cal}}{\text{g}}$$

EX02:

$$\begin{array}{l} 1) \text{Etat (1)} \longrightarrow \text{Etat (2)} \\ \left\{ \begin{array}{l} V_1 = 1 \text{ m}^3 \\ P_1 = 1 \text{ atm} \\ T_1 = 273,15 \end{array} \right. \quad \left\{ \begin{array}{l} V_2 \\ P_2 = 20 \text{ bars} \\ T_2 = T_1 \end{array} \right. \end{array}$$

$$a) V_2 = ?$$

$PV = \text{cte}$, loi de Boyle-Mariotte

$$P_1 V_1 = P_2 V_2 \Rightarrow V_2 = \frac{P_1 V_1}{P_2} \Rightarrow V_2 = \frac{1,013 \times 10^5 \times 1}{20 \times 1,013 \times 10^5} = 0,05 \text{ m}^3 \Rightarrow V_2 = 50 \text{ l}$$

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$$b) W = - \int P_{ext} dV ; P_{ext} = P_{gaz} = \frac{nRT}{V} \text{ (transformation reversible)}$$

$$W = - \int nRT \frac{dV}{V} \text{ avec } T = T_1 = T_2 = \text{cte}$$

$$W = -nRT_1 \ln \frac{V_2}{V_1} = -P_1 V_1 \ln \frac{V_2}{V_1}$$

$$W = 1013 \cdot 10^5 \times 1 \ln \frac{1}{0,05} = 303,5 \text{ KJ}$$

on applique le 1er principe $\Delta U = W + Q$

compression isotherme $\Delta U = 0$ ($\Delta T = 0$)

$$W + Q = 0 \Rightarrow Q = -W \Rightarrow Q = -303,5 \text{ KJ}$$

2) Etat (2) $\longrightarrow$ Etat (3)

$$\left\{ \begin{array}{l} P_2 \\ V_2 \\ T_2 \end{array} \right\} \quad \left\{ \begin{array}{l} P_3 \\ V_3 \\ T_3 \end{array} \right\}$$

$$a) V_3 = ? \quad T_3 = ?$$

$PV^\gamma = \text{cte}$ (loi de Laplace)

$$P_2 V_2^\gamma = P_3 V_3^\gamma \Rightarrow \frac{P_2}{P_3} = \left(\frac{V_3}{V_2} \right)^\gamma \Rightarrow V_3 = V_2 \left(\frac{P_2}{P_3} \right)^{1/\gamma}$$

$$\Rightarrow V_3 = 0,05 \left(\frac{20}{1} \right)^{1/1,4} \Rightarrow V_3 = 0,425 \text{ m}^3 = 425 \text{ L}$$

$$P_3 V_3 = nRT_3 \Rightarrow T_3 = \frac{P_3 V_3}{nR} = \frac{P_3 V_3}{P_1 V_1} T_1$$

$$T_3 = \frac{1,013 \cdot 10^5 \times 0,425 \times 293}{1,013 \cdot 10^5 \times 1} = 116,025^\circ \text{C} = -156,97^\circ \text{K}$$

$$nRT_1 = P_1 V_1 \Rightarrow nR = \frac{P_1 V_1}{T_1}$$

$$b) W = - \int_{V_2}^{V_3} P_{ext} dV ; P_{ext} = P_{gaz}$$

$$pV^\gamma = \text{cte} \rightarrow p = \frac{c}{V^\gamma} \quad (pV^\gamma = p_2V_2^\gamma = p_3V_3^\gamma = \text{cte})$$

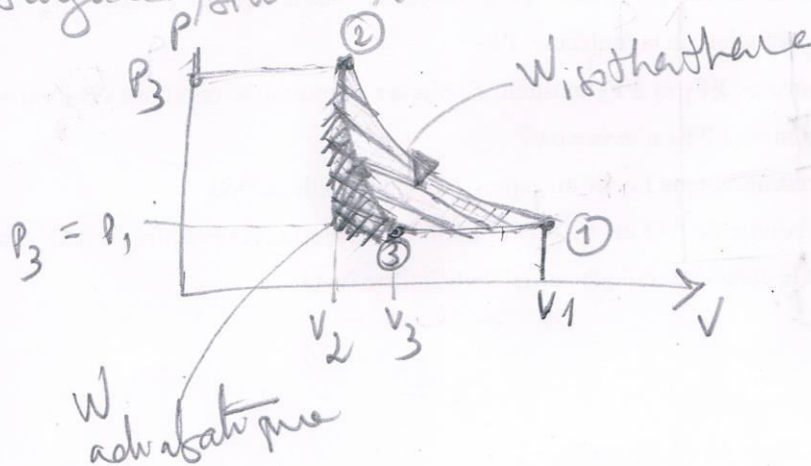
$$W = - \int_{V_2}^{V_3} p_2 V_2^\gamma \frac{dV}{V^\gamma} = - p_2 V_2^\gamma \int_{V_2}^{V_3} \frac{1}{V^\gamma} dV$$

$$W = - p_2 V_2^\gamma \left(\frac{1}{1-\gamma} \right) \left[V_3^{1-\gamma} - V_2^{1-\gamma} \right] = - p_2 V_2^\gamma \left(\frac{1}{1-\gamma} \right) \left(V_3^{1-\gamma} - V_2^{1-\gamma} \right)$$

$$W = 20 \cdot 10^5 (0,05)^{1,4} = \frac{1}{(1-1,4)} \left[(0,05)^{1-1,4} - (0,025)^{1-1,4} \right]$$

$$W = -145,66 \text{ KJ}$$

Le gaz travaille moins qu'avant, la surface située sous la courbe adiabatique est plus faible que la surface située sous l'isotherme.



Exo 3:

Compression isotherme

$$\begin{cases} A \\ p_A = 1 \text{ bar} \\ V_A = ? \\ T_A = 300 \text{ K} \end{cases}$$

$$\begin{cases} B \\ p_B \\ V_B \\ T_B = T_A \end{cases}$$

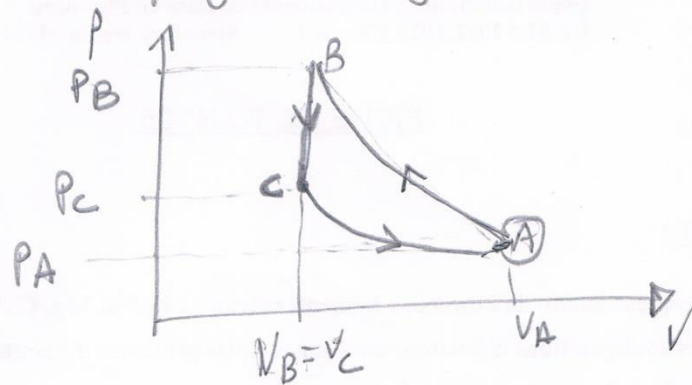
échauffement isochore $V = \text{cte} (V_B = V_C)$

$$\begin{cases} C \\ p_C \\ V_C = V_B \\ T_C \end{cases}$$

détente adiab.

$$\begin{cases} A \\ p_A \\ T_A \\ V_A \\ pV^\gamma = \text{cte} \end{cases}$$

Diagramme de Clapeyron ($P=f(V)$) qualitativement



Etat (A)

$$P_A V_A = n R T_A \Rightarrow V_A = \frac{n R T_A}{P_A}$$

$$\Rightarrow V_A = \frac{1 \times 8,314 \times 300}{10^5} = 0,0249 \text{ m}^3 = 24,9 \text{ L}$$

Etat (B) isotherme $T_B = T_A = 300^\circ \text{K}$

$$\frac{V_A}{V_B} = 10 \Rightarrow V_B = \frac{V_A}{10} = 2,49 \text{ L}$$

$$P_B V_B = n R T_B \Rightarrow P_B = \frac{n R T_B}{V_B}$$

$$P_B = \frac{1 \times 8,314 \times 300}{2,49 \times 10^{-3}} = 9,9768 \times 10^5 \text{ Pa} \approx \boxed{10 \text{ bars}}$$

Etat (C) isochore $V_C = V_B = 2,49 \text{ L}$

$$P_C = ? \quad T_C = ?$$

C $\rightarrow$ A adiabatique reversible $P V^\gamma = \text{cte}$

$$P_A V_A^\gamma = P_C V_C^\gamma \Rightarrow P_C = P_A \left(\frac{V_A}{V_C} \right)^\gamma$$

$$P_C = 1 \times \left(\frac{24,9}{2,49} \right)^{1,4} = 25,1 \text{ bars}$$

$$P_C V_C = n R T_C \Rightarrow T_C = \frac{P_C V_C}{n R} = \frac{25,1 \times 10^5 \times 2,49 \times 10^{-3}}{1 \times 8,314} = 751,7^\circ \text{K}$$

$$T_A V_A^{\gamma-1} = T_C V_C^{\gamma-1} \Rightarrow T_C = T_A \left(\frac{V_A}{V_C} \right)^{\gamma-1} = 300 \left(\frac{24,9}{2,49} \right)^{1,4-1} = 751,7^\circ \text{K}$$

	A	B	C
P(bar)	1	10	25,1
v(l)	2,49	2,49	2,49
TOK	300	300	751,7

3) A → B compression isotherme

$$W_{AB} = - \int_{V_A}^{V_B} P_{ext} dV = - \int_{V_A}^{V_B} nRT_A \frac{dV}{V} = -nRT_A \ln \frac{V_B}{V_A} = nRT_A \ln \frac{P_B}{P_A}$$

$$W_{AB} = 1 \times 8,314 \times 300 \ln \frac{10}{1} = +5743,17$$

$$\Delta U = 0 \quad (T = \text{cte}) \Rightarrow W_{AB} + Q_{AB} = 0 \Rightarrow Q_{AB} = -W_{AB} = -5743,17$$

il s'agit d'une compression isotherme, le système reçoit du travail $W > 0$

B → C échauffement isochore ($V = \text{cte}$)

$$W = - \int P_{ext} dV \quad (dV = 0) \Rightarrow \boxed{W_{BC} = 0}$$

$$\Delta U_{BC} = Q_{BC}$$

$$\Delta U_{BC} = \int_{T_B}^{T_C} nC_V dT = nC_V(T_C - T_B)$$

$$\begin{cases} C_P - C_V = R \\ \frac{C_P}{C_V} = \gamma \end{cases} \Rightarrow \begin{cases} C_V = \frac{R}{\gamma - 1} \\ C_P = \frac{\gamma}{\gamma - 1} R \end{cases}$$

$$\Rightarrow \Delta U = \frac{nR}{\gamma - 1} (T_C - T_B)$$

$$\Delta U_{BC} = Q_{BC} = \frac{1,8 \times 8,314}{1,4 - 1} (751,7 - 300) = 9388,587$$

C → A détente adiabatique

$$Q_{CA} = 0$$

$$\Delta U_{CA} = W_{CA}$$

$$\Delta U_{CA} = W_{CA} = \int_{T_C}^{T_A} n C_V dT = n C_V (T_A - T_C)$$

$$\Delta U_{CA} = \frac{nR}{\gamma - 1} (T_A - T_C)$$

$$\Delta U_{CA} = \frac{1 \times 8,314}{1,4 - 1} (300 - 751,7)$$

$$\Delta U_{CA} = W_{CA} = -9388,58 \text{ J}$$

Il s'agit d'une détente, le gaz fournit du travail au milieu extérieur. $W < 0$

Calcul de ΔH : $\Delta H = n C_p dT$

$$A \rightarrow B \quad \Delta H_{AB} = 0 \quad (T = \text{cte})$$

$$B \rightarrow C \quad \Delta H_{BC} = \int_{T_B}^{T_C} n C_p dT = nR \frac{\gamma}{\gamma - 1} (T_C - T_B)$$

$$\Delta H_{BC} = 1 \times 8,314 \times \frac{1,4}{1,4 - 1} (751,7 - 300) = 13144 \text{ J}$$

$$C \rightarrow A: \quad \Delta H_{CA} = \int_{T_C}^{T_A} n C_p dT = nR \frac{\gamma}{\gamma - 1} (T_A - T_C)$$

$$\Delta H = 1 \times 8,314 \times \frac{1,4}{1,4 - 1} (300 - 751,7) = -13144 \text{ J}$$

pergez

	A → B	B → C	C → A
$w(\tau)$	5743,1	0	-9388,58
$q(\tau)$	-5743,1	9388,58	0
$\Delta u(\tau)$	0	9388,58	-9388,58
$\Delta H(\tau)$	0	13144	-13144

$$Q_{\text{cycle}} = Q_{AB} + Q_{BC} + Q_{CA} \\ = -5743,1 + 9388,58 + 0 = +3645,48 \text{ J}$$

$$W_{\text{cycle}} = W_{AB} + W_{BC} + W_{CA} \\ = 574,1 + 0 - 9388,58 = -3645,48 \text{ J}$$

$$\Delta U_{\text{cycle}} = Q_{\text{cycle}} + W_{\text{cycle}} = +3645,48 - 3645,48 = 0$$

$W_{\text{cycle}} = -3645,48 \text{ J} < 0$ Il s'agit d'un cycle moteur, le système fournit du travail au milieu extérieur